



HVLS FANS

ASHRAE Philadelphia & New Jersey
April 21, 2021

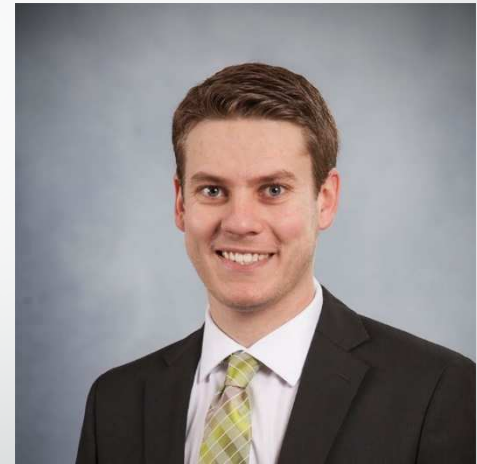


Housekeeping Guidelines

- Audience members are muted during the presentation.
- Questions can be submitted using the chat function during the presentation. They will be answered via chat or by the presenter at the end of the presentation.

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HVLS Fan Design, Application, & Specification



Learning Objectives

- Identify the function and application of HVLS fans
- Describe the primary fan design considerations for specifying HVLS fans
- Understand HVLS fan performance testing, data, and industry standards
- Explain the criteria and processes used to make appropriate HVLS fan selections

What is an HVLS Fan?

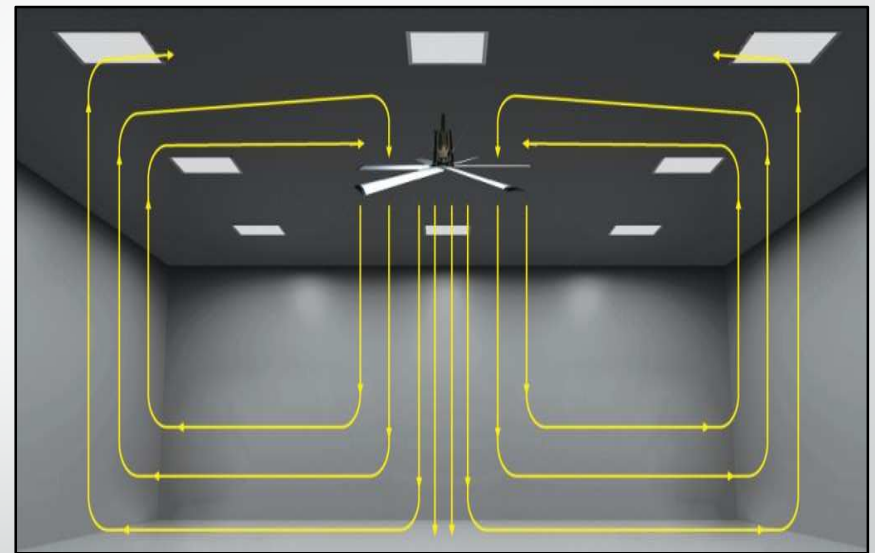


HVLS = High Volume Low Speed

Large diameter ceiling fans designed to circulate high volumes of air using low operational speeds

HVLS Fan Operating Principles

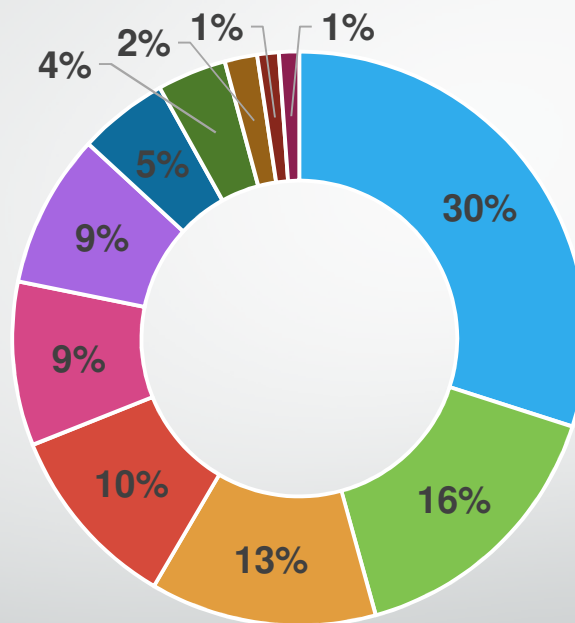
- High volume air movement
 - Large diameter results in large column of air being displaced
- Low operational speeds
 - Gentle air movement with minimal turbulence
 - Low sound levels
 - Less horsepower required to operate fan
- Large area of effect
 - Coanda effect causes air to cling to surfaces and entrain surrounding air
 - Large air mass capable of traveling long distances





HVLS Fan Market & Applications

Construction Market Distribution



Industrial
Government

Educational
Agricultural

Retail
Community

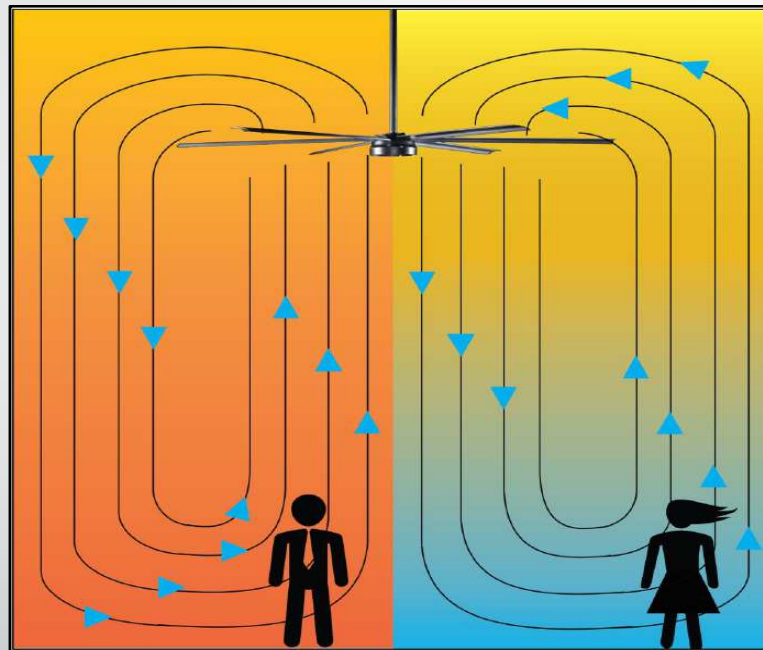
Commercial
Civil

Applications

Thermal Comfort & Energy Efficiency

Winter

- Destratification
- Reduce heat loss through roof
- Save up to 25% on heating costs



Summer

- Air circulation & evaporative cooling
- Improve occupant efficiency
- Save up to 30% on cooling costs

Applications

Safety & Inventory Integrity



247,120

Occupational injuries and illnesses — related to falls, slips and trips — that led to private industry employees missing work in 2014.

PROBLEMS CAUSED BY SWEATING SLAB SYNDROME



INJURY RISK

Employees are more prone to slipping and falling.



FORKLIFT FOLLY

Product transporters become more difficult to maneuver, and braking can fail.



INVENTORY DAMAGE

Metal products are prone to corrosion; other products could deteriorate, and packaging can dampen.



LUBRICANT LEAKS

The condensation on motorized machines can mix with chemical lubricants and leak.



EFFLORESCENCE

Moisture prompts sulfates in the concrete to rise to the surface, resulting in salty deposits on the floor.



BUILDING RISKS

Moisture promotes mold growth.

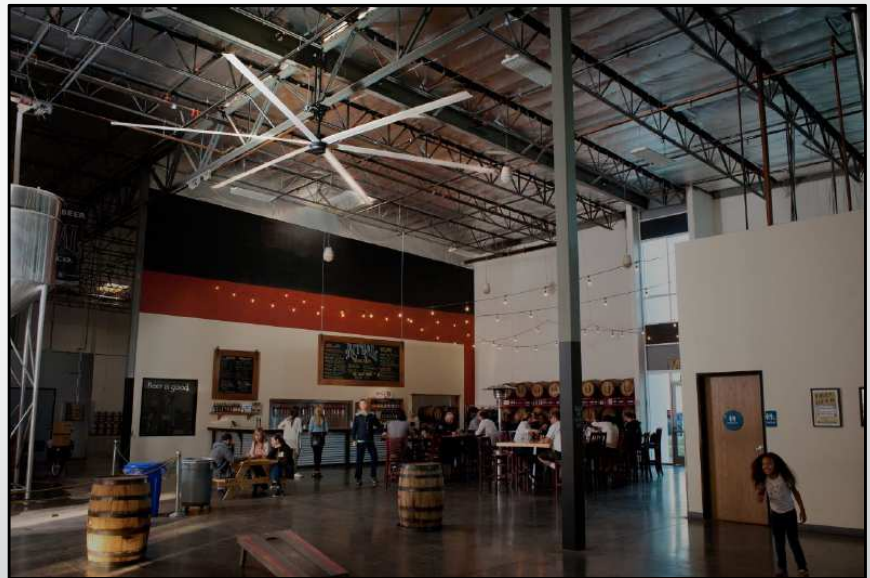
Applications

Pest Deterrence



Applications

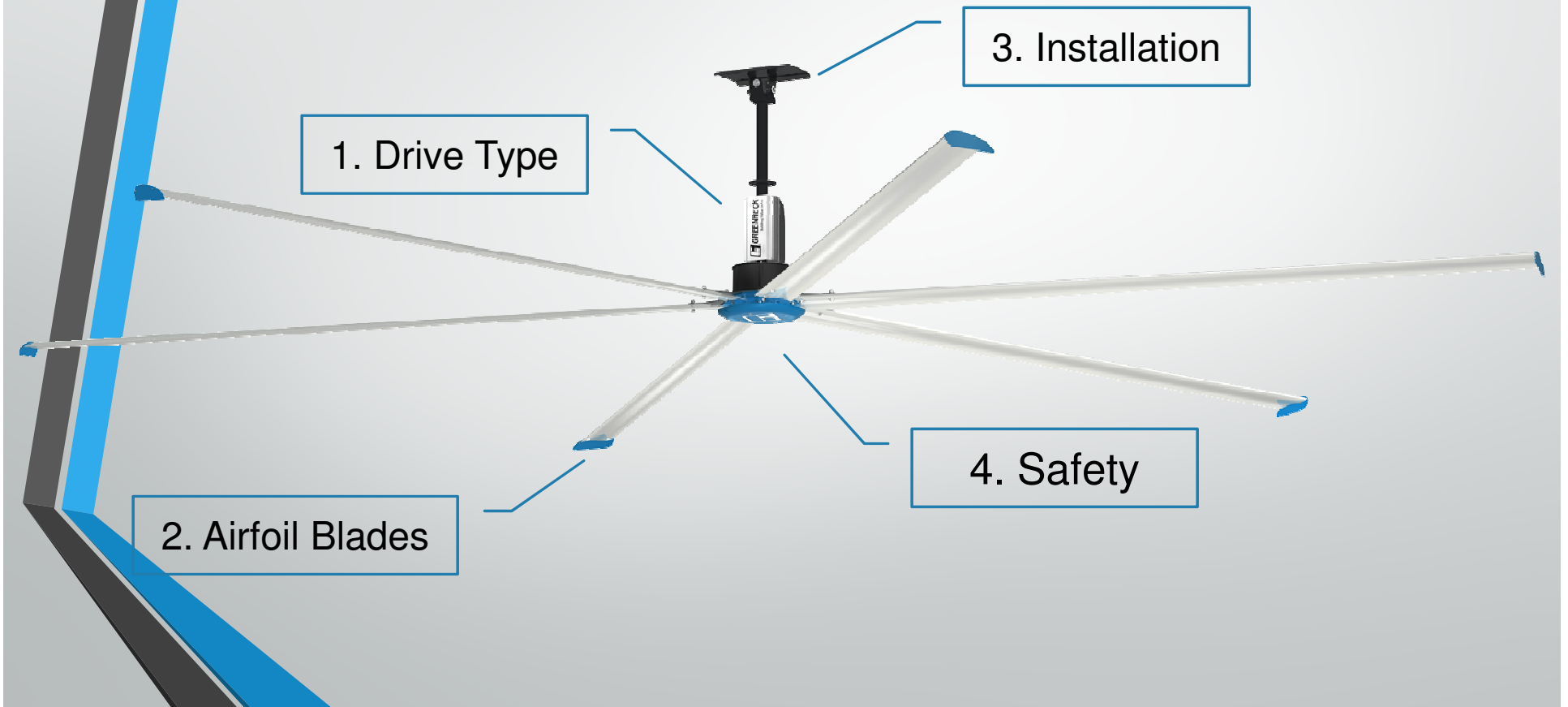
Architectural Design







HVLS Fan Design Considerations

Fan Design Considerations



Drive Type

		Description	Pros	Cons
Gearbox		High RPM motor with gear system that reduces speed to maximize torque ($P = \tau * \text{RPM}$)	<ul style="list-style-type: none"> • Motors more readily available • Lower first-cost • Easily applied to any diameter 	<ul style="list-style-type: none"> • More maintenance (oil changes) • Physically larger/heavier • Efficiency losses • Can be noisy
		Low RPM motor designed for high continuous torque	<ul style="list-style-type: none"> • Little to no maintenance • Compact design • High efficiency • Quiet 	<ul style="list-style-type: none"> • Limited motor availability • Higher first-cost • Not always available for large diameters



Airfoil Blades

- Factors to consider
 - Blade count
 - More blades not necessarily better
 - 5 or 6 blades = best balance of airflow & efficiency

$$P = \tau * RPM$$

	3-Blade Fan	6-Blade Fan
Motor Power	500 W	500 W
Max RPM	86	69
Max CFM	124,500	128,100

Airfoil Blades

- Factors to consider
 - Blade count
 - More blades not necessarily better
 - 5 or 6 blades = best balance of airflow & efficiency
 - Blade deflection
 - Blade structure and materials vary
 - Critical for preventing unsafe operation

24 ft. Diameter	Static Blade Deflection
Manufacturer A	3.7 in.
Manufacturer B	9.0 in.

Fan Installation

- Factors to consider
 - Installation location
 - Accessibility, clearance, structure, etc.



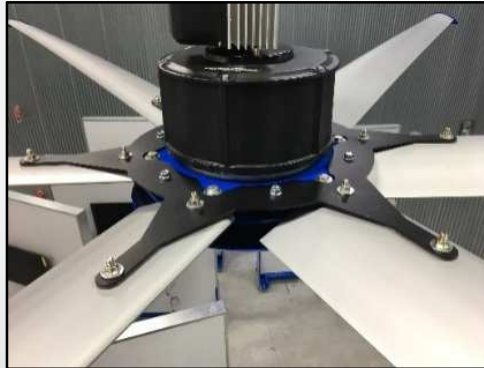
Fan Installation

- Factors to consider
 - Installation location
 - Accessibility, clearance, structure, etc.
 - Fan weight
 - Lighter weight = lower installed costs

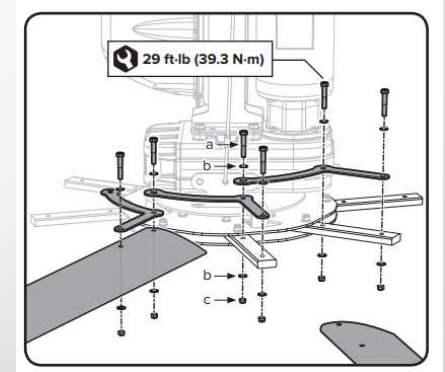
24 ft. Diameter	Weight (lbs.)
Manufacturer A	214
Manufacturer B	231
Manufacturer C	239
Manufacturer D	300
Manufacturer E	347

Product Safety

- Factors to consider
 - Mechanical safety systems
 - Factory-installed systems prevent installation problems

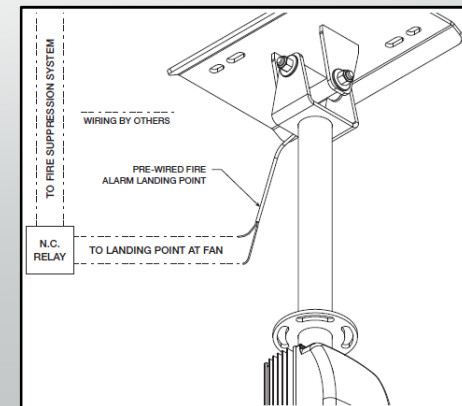


VS.



Product Safety

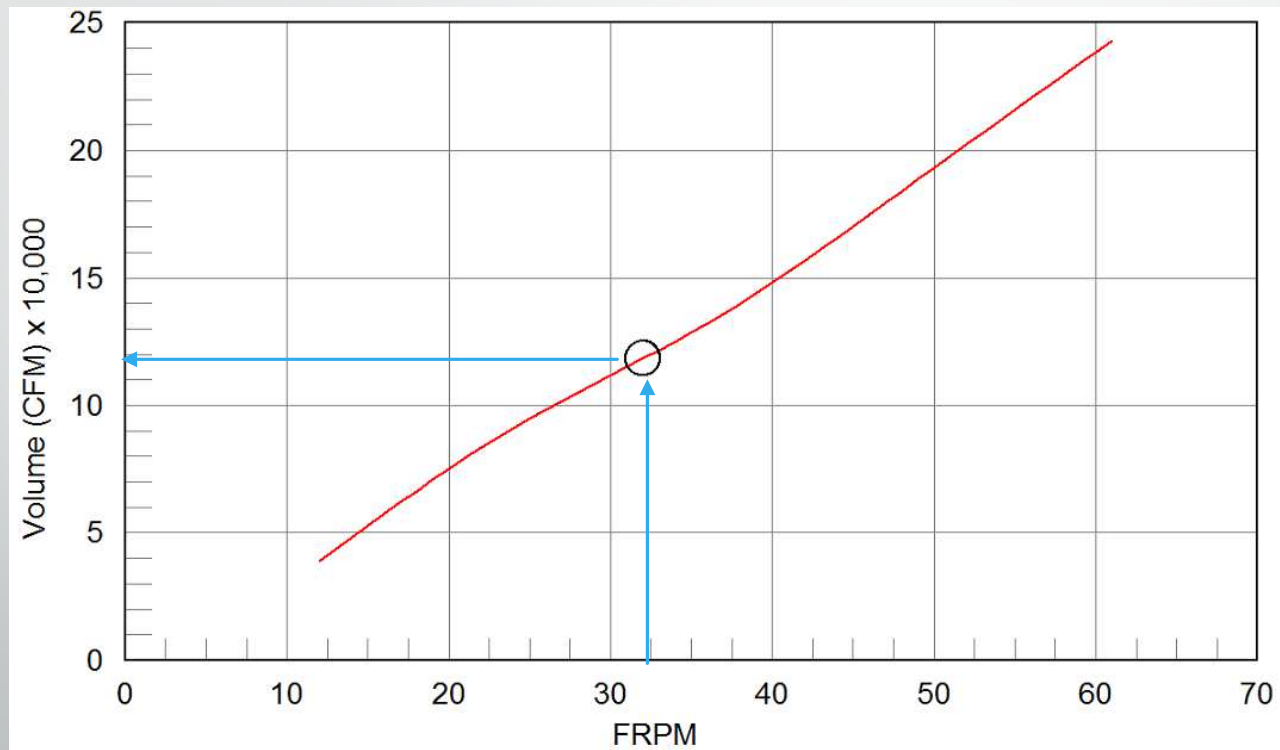
- Factors to consider
 - Mechanical safety systems
 - Factory-installed systems prevent installation problems
 - Fire system integration
 - NFPA 13 requires:
 1. Maximum fan diameter shall be 24 ft.
 2. Fan shall be centered between 4 sprinklers
 3. Vertical clearance to sprinkler deflector shall be minimum of 3 ft.
 4. Fans shall be interlocked to shut down upon receiving fire alarm
 - Factory-supplied parts simplify installation





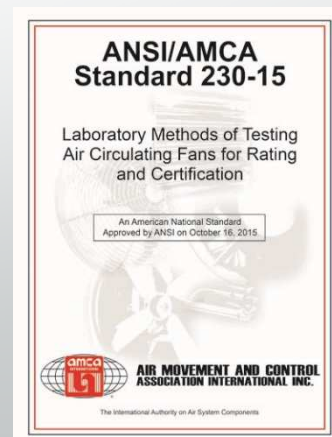
HVLS Fan Performance

Fan Curves



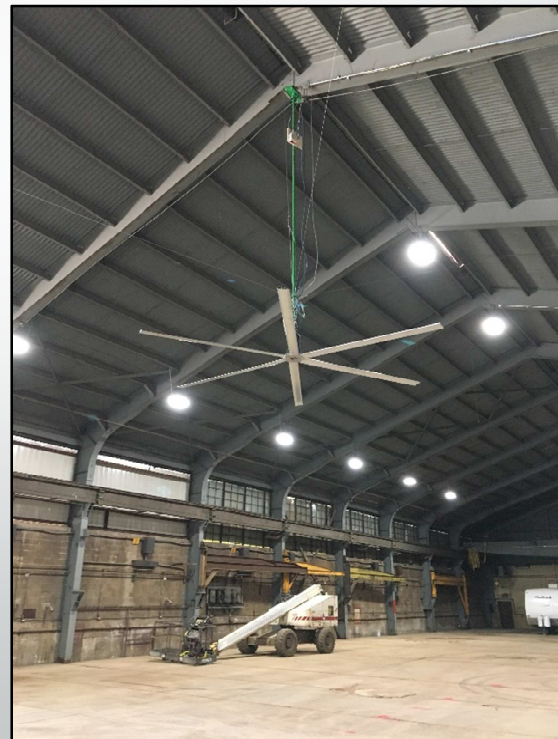
AMCA International

- Air Movement and Control Association
 - Independent third party verification
 - International Certified Ratings Program
 - Guaranteed performance as stated with AMCA seal



HVLS Fan Testing

- Cannot be tested using traditional air chamber
 - Requires large open area with high ceilings
- Airflow is not measured directly
 - Measure thrust generated by fan using a load cell
- Power determined in two ways
 - Measure torque and RPM to calculate mechanical power
 - Measure input electrical power using power meter



Importance of AMCA Certification

- Guaranteed performance as stated with AMCA seal
- History of inaccurate performance data in HVLS industry
 - Previous performance calculations incorrectly included $\sqrt{2}$ correction factor resulting in ~30% higher CFM values
 - Many manufacturers continue to publish data calculated using these equations
 - No driving force for correcting published data until recently

Previous Calcs.

$$Q_0 = 340.3 \sqrt{\frac{2AF_t}{\rho_{std}}}$$

$$F_t = 37.0$$

$$A = 113 \text{ ft}^2$$

$$Q_0 = 113,664 \text{ CFM}$$

AMCA 230-15

$$Q_0 = 340.3 \sqrt{\frac{AF_t}{\rho_{std}}}$$

$$F_t = 37.0$$

$$A = 113 \text{ ft}^2$$

$$Q_0 = 80,365 \text{ CFM}$$

DOE Efficiency Legislation

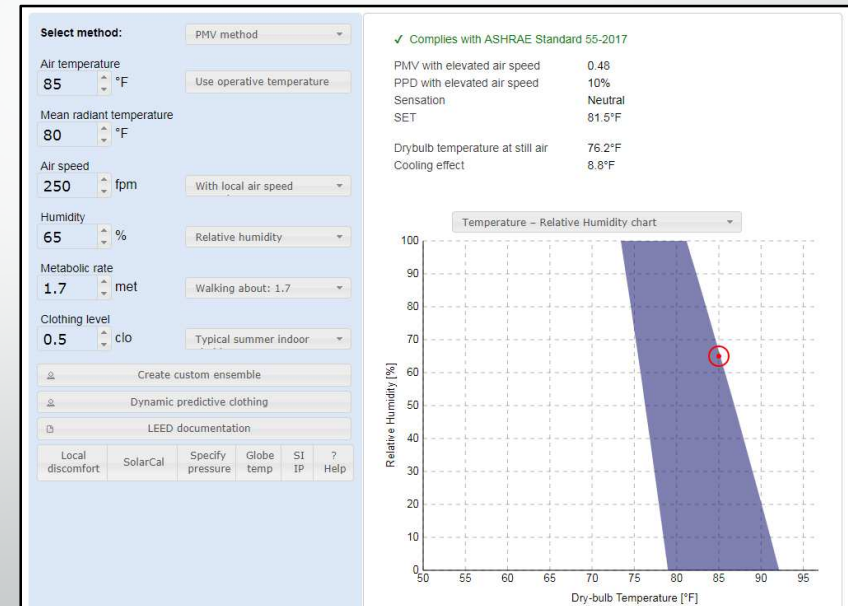
- Department of Energy (DOE) developed minimum efficiency standards for ceiling fans
- Adopted into national law and enforceable starting 2020
- DOE rule is in sync with AMCA 230!
 - Will require manufacturers to correct published performance data
- Efficiency minimums intended to obsolete least efficient products (~10%)

Therefore, DOE requires all large-diameter ceiling fans to be tested according to AMCA 230–15, but with the modification that the number of speeds to be tested is as set forth in Table 2.

Fan Diameter	Min. Efficiency (CFM/W)
8	57
10	79
12	101
14	123
16	145
18	167
20	188
24	232

Other Performance Considerations

- Air velocity in occupied zone
 - Studies have shown employee efficiency and comfort benefits of higher air velocity when temperature/humidity are high
 - ASHRAE 55 establishes methodology for quantifying effect of air velocity on thermal comfort



Other Performance Considerations

- Sound

- Typically published as total dBA (A-weighted sound pressure including fan & air noise)
- No test standards so test procedures vary
- Commonly shown at distance 20 ft. from fan, measured 5 ft. above floor



5 ft.

20 ft.

Performance is Critical!

“Aren’t all 24 ft. fans the same?”

	Manufacturer A	Manufacturer B	Difference
AMCA Certified	✓	-	-
Max CFM	243,000	217,000	+12%
Efficiency (CFM/W)	249	217	+15%
dBA at Max RPM	50	55	-9%
Coverage Area (Sq. Ft.)	23,700	20,736	+14%

Not all HVLS fans are created equal!



Performance Specification Language

Ensure accurate performance on HVLS fans by specifying AMCA!

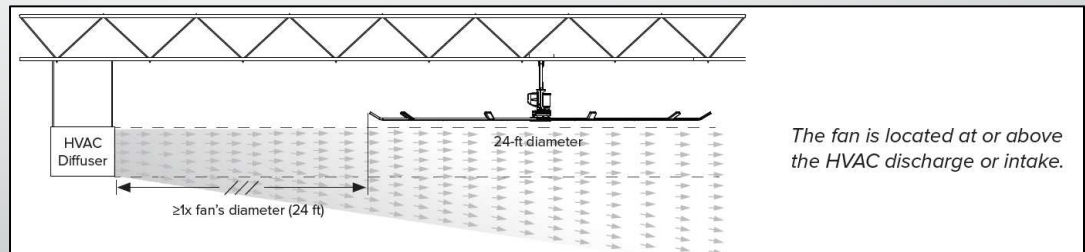
“Performance ratings for HVLS fans shall conform to AMCA standard 211. Fans must be tested in accordance with ANSI/AMCA Standard 230-15 in an AMCA accredited laboratory. Fans shall be certified to bear the AMCA Seal for Circulating Fan Performance.”



HVLS Fan Selection & Specification

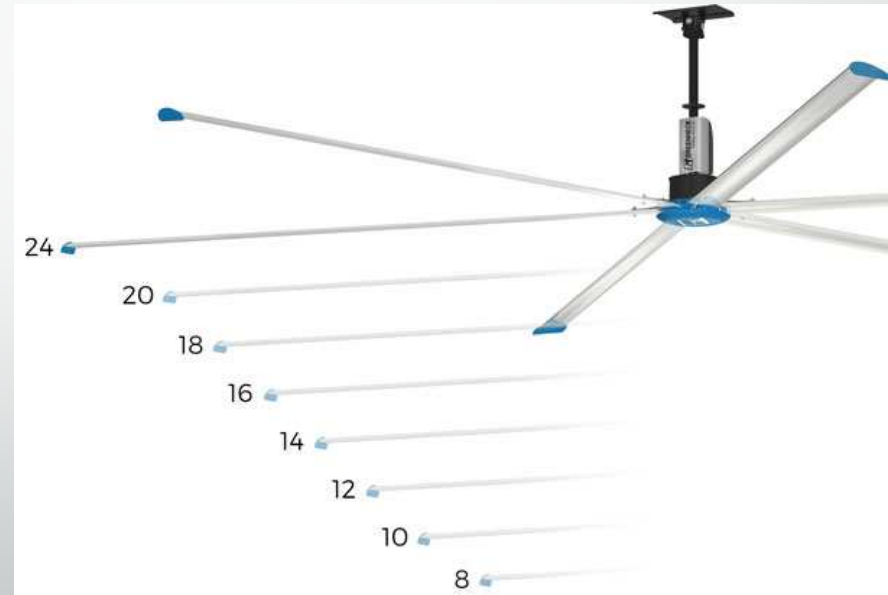
Selection Considerations

- Building type & application
- Fan design
- Fan performance (airflow, efficiency, sound, air velocity)
- Installation location
 - Accessibility, structural support, etc.
- Airflow obstructions
 - Anything that disrupts air movement
 - Walls, furniture, equipment, racking, etc.
- Clearance requirements
 - Clearance to physical obstructions
 - Clearance to HVAC inlets/outlets



Selection Process

- Processes vary among design professionals
- Two primary methods
 - Size-based selection
 - Performance-based selection
- Size-based method is most commonly used today



Size-Based Selection

• Process

- Utilize published coverage area or fan spacing values to identify quantity and size of fans that physically fit space

• Pros

- Easy and fast
- Generally a “safe” design

• Cons

- No performance considerations
- No data to support design decisions
- Typically over-designs systems leading to higher first-cost

- Space the fans at a center-to-center distance that is at least 2.5x the fan diameter.

PERFORMANCE

Max Speed	75 RPM
Recommended Spacing*	105 ft [32 m]
Max Affected Area	20,000 ft ² [1,858 m ²]

Open Area Fan Requirements*

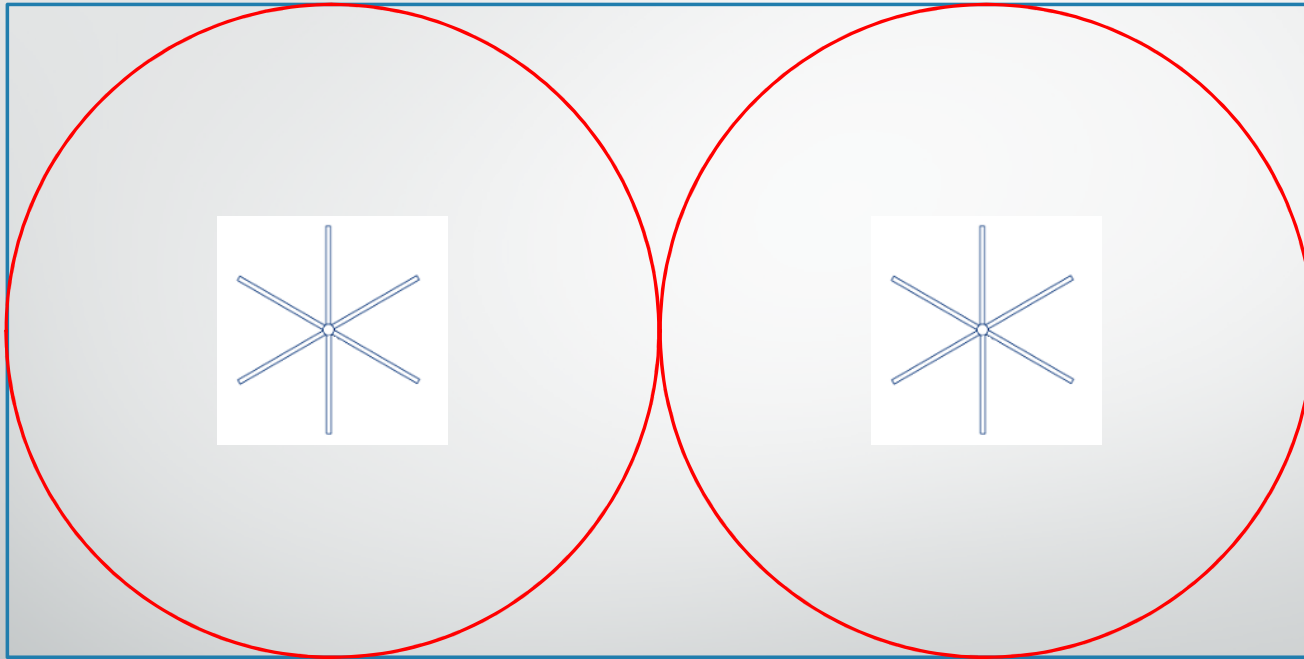
Length	Width					
		100'	200'	300'	400'	500'
	100'	1	1 or 2	2	2 or 3	3
	200'	1 or 2	2	2 or 3	3 or 4	4 or 5
	300'	2	2 or 3	4 or 5		
	400'	2 or 3	3 or 4			
	500'	3	4 or 5			

*Grid and chart based on 30' ceiling heights and 24' diameter fans. Fans in open areas may cover up to 85 feet from the fan's center in all directions.

Size-Based Selection

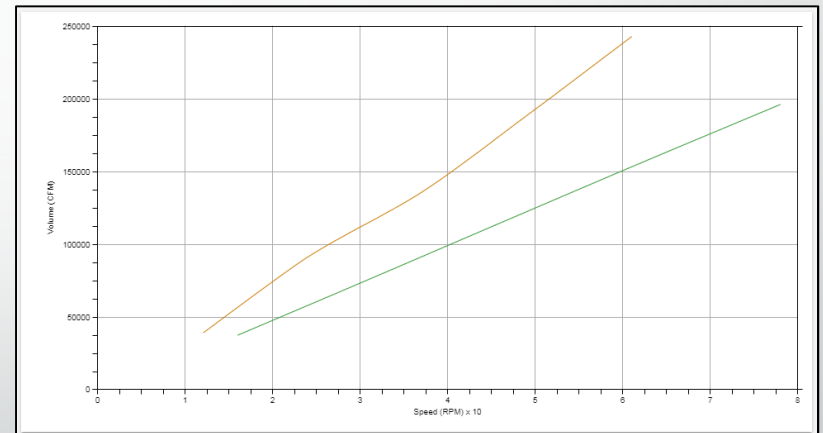
100 ft.

200 ft.



Performance-Based Selection

- Process
 - Utilize performance data to identify size and quantity of fans that deliver correct performance
 - Based on industry standards (AMCA, ASHRAE, etc.)
- Pros
 - Better system design that balances cost & performance
 - Data to support design decisions
- Cons
 - Few manufacturers that publish data
 - Software not always public
 - Can be more time-consuming



Performance-Based Selection

Selection Method
By Performance

Space Length (ft)
100

Space Height (ft)
20

Available Sizes
Recommended

Sizing Method
Comfort Coolin

Space Width (ft)
200

Minutes / Air Rotation
3

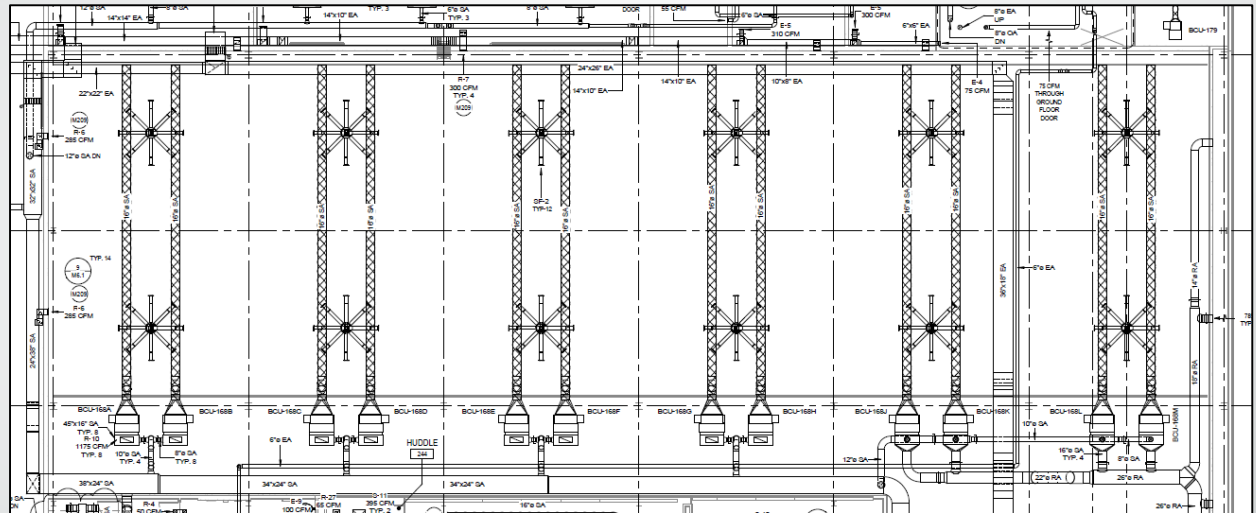
Selection Example

- Manufacturing facility in northern climate
- Fans needed for both destratification and cooling applications
- 185' x 68' open space with 25' ceiling

DARE
TO **compare**

Size-Based Selection

- Qty. (12) 10 ft. fans specified using manufacturer spacing data
- No performance data to populate schedule



FAN SCHEDULE

TYPE	AIR CAPACITY			MIN DIAMETER	TSP	ESP	FAN RPM	OCTAVE BANDS, MAX DUTY POINT, MAX PWL (DB RE 10 ⁴ -(12)) W								MOTOR DATA				
	MIN	DESIGN	MAX					1 (63 HZ)	2 (125 HZ)	3 (250 HZ)	4 (500 HZ)	5 (1,000 HZ)	6 (2,000 HZ)	7 (4,000 HZ)	8 (8,000 HZ)	BHP	HP	MAX RPM	VOLTAGE	PHASE
AXIAL	11000	11000	11000	3' - 0"	1.28 in-wg	1.13 in-wg	1456	87	93	92	90	89	86	83	80	4.26	7.5	1760	460 V	3
PLENUM	11000	11000	11000	2' - 6"	4.87 in-wg	4.78 in-wg	1360	93	92	94	97	96	92	88	82	12.88	20	1760	460 V	3
AIRFOIL CEILING	0	0	0	10' - 0"	0.00 in-wg	0.00 in-wg	107	0	0	0	0	0	0	0	0	0	0	107	208 V	1

Performance-Based Selection


- Qty. (6) 10 ft. fans
 - Based on required airflow and air velocity to meet project requirements
 - Significant first-cost savings compared to size-based selection
- Other selection options available
 - (1) 20 ft. fan
 - (3) 14 ft. fans
- Data to populate schedule including:
 - CFM per fan
 - Affected area per fan
 - Required operating RPM
 - Power consumption per fan
 - Fan efficiency
 - Total dBA per fan

Relative Cost	Quantity	Total Operating Cost/Year (USD)	Impeller Diameter (in.)	Actual Volume Per Fan (CFM)	Affected Area Per Fan (ft2)	Min. Fan Spacing (ft)	Fan Spacing (ft)	Avg. Air Speed (ft/min)	Max Avg. Air Speed (ft/min)	Direction of Operation	Fan Speed (RPM)	Max Fan Speed (RPM)	Motor Size (W)	Standby Power Per Fan (W)	208/60/3 Input Watts Per Fan	Integrated Efficiency Per Fan (CFM/W)	Total dBA Per Fan
5.56	6	13	120	17,654	2,097	30	52	114	282	Forward	55	136	500	11	54	128	30

Summary

- HVLS fans are highly engineered products that can provide significant value to HVAC and ventilation system designs in any facility
- Not all HVLS fans are created equal, so it is critical to be informed about product and performance differences
- Design professionals need to account for a variety of factors to achieve cost-effective systems that meet customer performance requirements





Appendix

History of

- Invented in the late 1990's
- Designed for agricultural applications
 - Livestock comfort & health
- Transitioned into commercial & industrial applications in the early 2000's
 - Similar benefits for building occupants!
 - Exponential growth rate over ~10 years



- International test standard for consistent performance testing of circulating fans

AMCA 230

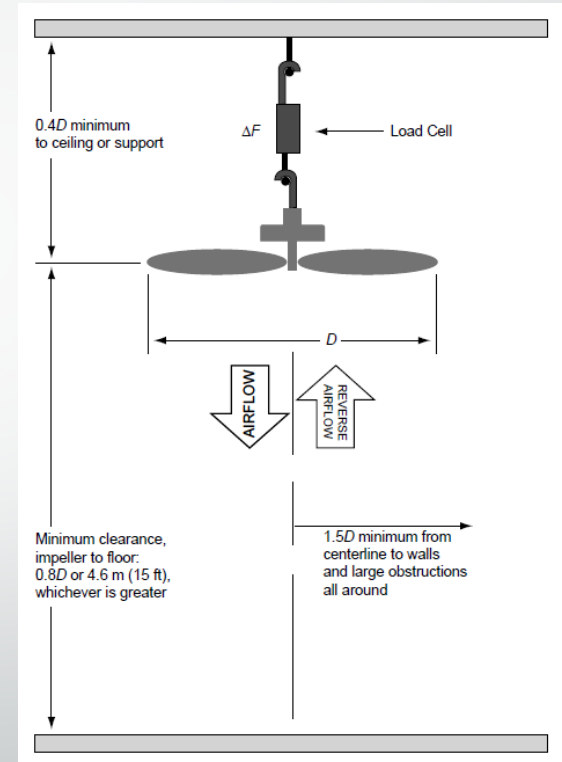
- Part of CRP program for AMCA certification of air performance data

- Test setup for 24 ft. HVLS fan:

- Minimum 72' x 72' x 29' space
- Fan installed at least ~20' above floor
- Load cell installed between fan and ceiling mount
- Power meter measuring input electrical power
- Optical tachometer measuring RPM
- Wet-bulb, dry-bulb, and barometric pressure measurements for calculating air density
- No extraneous airflow (>50 FPM) in test area

- Test procedure

- Performance data recorded at 5 test speeds (20, 40, 60, 80, 100%) and standby mode



AMCA

- Airflow calculation

- Measure load differential
- Calculate thrust force at standard air density (70°F at sea level)
- Calculate airflow using thrust and area of the fan sweep (outlet area)

Thrust shall be calculated according to the following:
For Test Figures 1, 3A and 3B:

$$F_t = \Delta F \left(\frac{\rho_{std}}{\rho_0} \right) \quad \text{Eq. 9.4}$$

ΔF = Load differential, N (lbf)

ρ_0 = Ambient air density, kg/m³ (lbm/ft³)

ρ_{std} = Standard air density, 1.2 kg/m³ (0.075 lbm/ft³)

$$Q_0 = 340.3 \sqrt{\frac{AF_t}{\rho_{std}}} \quad \text{Eq. 9.6 I-P}$$

Q_0 = Airflow rate, m³/s (cfm)

F_t = Thrust, N (lb)

A = $\pi(D/2)^2$, m² (ft²)

ρ_{std} = Standard air density, 1.2 kg/m³ (0.075 lbm/ft³)

AMCA 230

- Airflow calculation
 - Measure load differential
 - Calculate thrust force at standard air density (70°F at sea level)
 - Calculate airflow using thrust and area of the fan sweep (outlet area)
- Fan efficiency calculation
 - Calculate airflow
 - Measure input electrical power in watts
 - Calculate efficiency in terms of airflow / watt

$$Eff_{circ} = \frac{Q_0}{W_E}$$

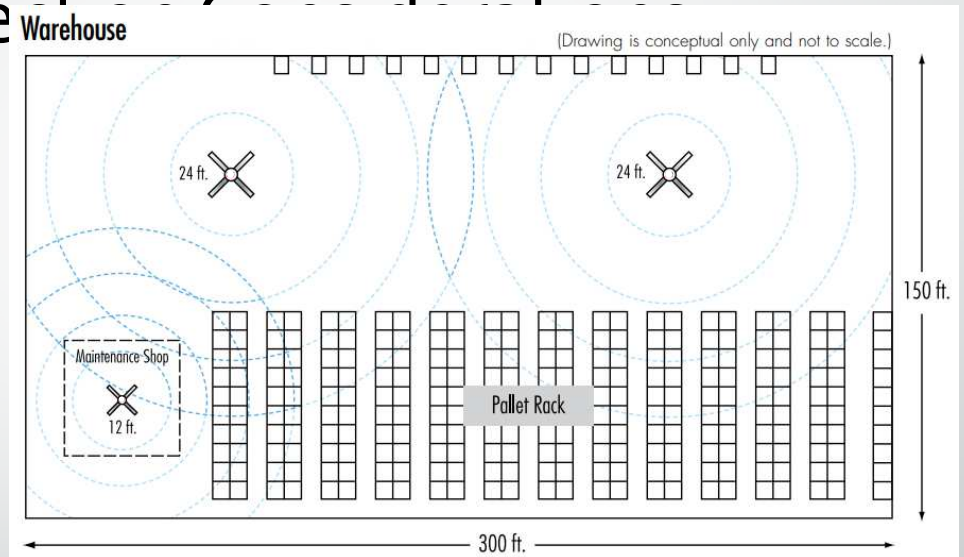
Eq. 9.8

Where:

Q_0 = Fan airflow rate m³/s (cfm)
 W_E = Electrical input power, watt

Additional Selection Criteria

- Impact on heating & cooling loads
 - Should design conditions be changed?
- Controls
 - Quantity
 - Location
 - Functionality





Thank you for your time.

Questions?



The mission of Greenheck is to be the market leader in the development, manufacture and worldwide sales of quality air moving, control and conditioning equipment with a total commitment to customer service.

